

34.21 A Brayton cycle with regeneration has a regenerator with 60% effectiveness. Air enters the compressor at $60^\circ F$ and 14.7psia and is compressed to 150psia . Air enters the turbine at $1400^\circ F$ and leaves at $600^\circ F$ and 14.7psia . Assuming the compressor and turbine have isentropic efficiencies of 100%, what is the change in enthalpy during combustion?

- A. $108 \frac{\text{Btu}}{\text{lb}}$
- B. $119 \frac{\text{Btu}}{\text{lb}}$
- C. $197 \frac{\text{Btu}}{\text{lb}}$
- D. $204 \frac{\text{Btu}}{\text{lb}}$

Refer to the schematic under **Brayton Cycle with Regeneration**. Since the compressor is isentropic, determine the temperature at State 2 for a **Constant Entropy Process**. Use absolute temperature i.e. Rankine.

$$\frac{T_2}{T_1} = \left(\frac{P_2}{P_1}\right)^{\left(\frac{k-1}{k}\right)}$$

$$T_2 = T_1 \left(\frac{P_2}{P_1}\right)^{\left(\frac{k-1}{k}\right)} = (520R) \left(\frac{150\text{psia}}{14.7\text{psia}}\right)^{\left(\frac{1.4-1}{1.4}\right)} = 1009.8R = 549.8^\circ F$$

Apply the effectiveness of the regenerator to determine the temperature at State 3, prior to the combustor.

$$\varepsilon = \frac{h_3 - h_2}{h_5 - h_2} = \frac{c_p(T_3 - T_2)}{c_p(T_5 - T_2)} = \frac{(T_3 - T_2)}{(T_5 - T_2)}$$

$$T_3 = T_2 + \varepsilon(T_5 - T_2) = 549.8^\circ F + 0.6(600^\circ F - 549.8^\circ F) = 579.9^\circ F$$

Calculate the change in enthalpy through the combustor.

$$h_4 - h_3 = c_p(T_4 - T_3) = \left(0.24 \frac{\text{Btu}}{\text{lb} \cdot ^\circ F}\right) (1400^\circ F - 579.9^\circ F) = 196.8 \frac{\text{Btu}}{\text{lb}}$$

Answer C